Modeling of Soft Robots

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1 Part A: Continuum formulation of the dynamics

2 Part B: Discrete formulation with piecewise constant deformation

3 Part C: Piecewise constant curvature as special case of the deformation space formulation

Part A: Continuum formulation of the dynamics



- Mathematical description:

Continuum mechanics





soft robotic arm: one dimension is dominant over the two others



soft robotic arm: one dimension is dominant over the two others





Modeling assumption

A soft robotic arm is seen as the **continuous assembly of 2D cross–sections** moving upon a **3D curve** according to **infinite rigid body transformations** which are defined by **distributed laws of internal deformations**.



Modeling strategy

We derive the kinematics of continuum robots from the evolution of a 3D curve in space

- using a geometric local frame approach
- with no approximation on kinematic variables
- including all the components of deformation



Position field

 $\alpha \in \mathbb{R} \mapsto \boldsymbol{H}(\alpha) = \mathcal{H}(\boldsymbol{R}(\alpha), \boldsymbol{u}(\alpha)) \in SE(3)$

- $\alpha \in \mathbb{R}$, material abscissa along the arm
- $\boldsymbol{u}(\alpha) \in \mathbb{R}^3$, position vector of the cross-section
- $\mathbf{R}(\alpha) = [\mathbf{t}(\alpha) \ \mathbf{n}(\alpha) \ \mathbf{b}(\alpha)] \in SO(3)$, rotation matrix of the cross-section, where $\mathbf{t}(\alpha) =$ unit tangent vector; $\mathbf{n}(\alpha) =$ unit normal vector; $\mathbf{b}(\alpha) =$ unit binormal vector.

with

SO(3) special Orthogonal group: the Lie group of the rotation matrices SE(3) special Euclidean group: the Lie group of the homogeneous matrices

Positions as Lie group elements

Change of reference frame = Euclidean transformation The space SE(3) of Euclidean transformations is a **Lie Group**

$$\mathbf{H} = \mathcal{H}(\mathbf{R}, \mathbf{x}) = \begin{bmatrix} \mathbf{R} & \mathbf{x} \\ \mathbf{0}_{1\times 3} & 1 \end{bmatrix} \in SE(3)$$
$$SE(3) = SO(3) \times \mathbb{R}^{3}$$

Lie group

Definition

A group (G, \cdot) is a set G of elements q together with a composition operation (\cdot) which satisfies the four axioms of:

- closure: the composition of two elements of the set yields an element of the set, i.e., $\forall q_1, q_2 \in G, q_1 \cdot q_2 = q_3 \in G$
- associativity: $q_1 \cdot (q_2 \cdot q_3) = (q_1 \cdot q_2) \cdot q_3$
- neutral element: there exists an element e of the set such that $q \cdot e = e \cdot q = q$
- inverse element: there exists an element q^{-1} of the set such that $q \cdot q^{-1} = q^{-1} \cdot q = e$

Definition

A Lie group is a continuous group for which the composition rule and the inverse are smooth

Lie group (cont'd)

Proposition

A <u>matrix Lie group</u> is a Lie group for which the composition rule is represented by the matrix product

 $\mathbf{R} \in SO(3)$, the special Orthogonal group $\mathbf{H} \in SE(3)$, the special Euclidean group

Deformation field

The homogeneous matrix H evolves along the material abscissa α according to the differential kinematic relationship

$$H'(\alpha) = H(\alpha)\widetilde{f}(\alpha)$$

where $\tilde{f}(\alpha)$ is a left invariant vector field.

$$\widetilde{\mathbf{f}} = \begin{bmatrix} \widetilde{\mathbf{f}_{\omega}} & \mathbf{f}_{u} \\ \mathbf{0}_{1 \times 3} & 1 \end{bmatrix} \in \mathfrak{se}(3)$$
$$\mathbf{f}_{u} \in \mathbb{R}^{3}$$
$$\widetilde{\mathbf{f}_{\omega}} = \begin{bmatrix} 0 & -f_{\omega,3} & f_{\omega,2} \\ f_{\omega,3} & 0 & -f_{\omega,1} \\ -f_{\omega,2} & f_{\omega,1} & 0 \end{bmatrix} \in \mathfrak{so}(3)$$

deformation twist

vector of linear deformations

skew-symmetric matrix of angular deformations

Deformations as Lie algebra elements

•
$$(\widetilde{\cdot})_{SO(3)} : \mathbb{R}^3 \to \mathfrak{so}(3)$$

• $(\widetilde{\cdot})_{SE(3)} : \mathbb{R}^6 \to \mathfrak{se}(3)$

with

 $\mathfrak{so}(3)$ Lie algebra associated to the Lie group SO(3) $\mathfrak{se}(3)$ Lie algebra associated to the Lie group SE(3) \Downarrow

 $\widetilde{\mathbf{f}}(lpha) \in \mathfrak{se}(3)$ deformation twist $\mathbf{f}(lpha) \in \mathbb{R}^6$ deformation vector (axial, shear, bending, torsion)

Lie derivatives

Derivative of a Lie group

The derivative of $q \in G$ with respect to $a \in \mathbb{R}$ reads

$$egin{array}{ll} d_{a}(q) &= q \widetilde{\mathbf{a}}_{L} \ &= \widetilde{\mathbf{a}}_{R} q \end{array}$$

where $\tilde{a}_L \in \mathfrak{g}$ and $\tilde{a}_R \in \mathfrak{g}$ are respectively called a left and right invariant vector field. These elements represent the Lie algebra associated to the Lie group.

Lie algebra

Definition

The Lie algebra $\mathfrak{g}(\mathfrak{se}(3))$ is the tangent space at the identity element of a Lie group $G(\mathbf{H})$.

 $d_a(H) = H\widetilde{a}$



Lie algebra (cont'd)

Proposition

The Lie algebra \mathfrak{g} is isomorphic to \mathbb{R}^k through the invertible linear map

$$\widetilde{(\cdot)}: \mathbb{R}^k o \mathfrak{g}, \qquad \mathbf{a} \in \mathbb{R}^k \mapsto \widetilde{\mathbf{a}} \in \mathfrak{g}$$

Lie algebra (cont'd)

 $SO(3) \quad \mathfrak{so}(3) \quad \mathbb{R}^3$ $SE(3) \quad \mathfrak{se}(3) \quad \mathbb{R}^6$

Lie algebra (cont'd)

$d_a(\mathbf{H}) = \mathbf{H}\widetilde{\mathbf{a}}$ Left invariant vector field on SE(3)= Invariant under a superimposed Euclidean transformation = Intrinsic quantity

Differential geometry of the 3D curve



Serret-Frenet Formulas

 $\begin{aligned} \mathbf{t}'(\alpha) &= \kappa(\alpha)\mathbf{n}(\alpha) \\ \mathbf{n}'(\alpha) &= -\kappa(\alpha)\mathbf{t}(\alpha) + \tau(\alpha)\mathbf{b}(\alpha) \\ \mathbf{b}'(\alpha) &= -\tau(\alpha)\mathbf{n}(\alpha) \\ \bullet & \kappa(\alpha) \text{ and } \tau(\alpha): \text{ curvature and torsion of the curve} \end{aligned}$

 $\mathbf{H}'(\alpha) = \mathbf{H}(\alpha) \widetilde{\mathbf{f}}(\alpha)$

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• $\widetilde{\mathbf{f}}(\alpha)$: deformation twist

Compatibility equations

 $\boldsymbol{\eta}'(\alpha) - \dot{\boldsymbol{f}}(\alpha) = \widehat{\boldsymbol{\eta}}(\alpha)\boldsymbol{f}(\alpha)$

Lie bracket

Definition

The Lie bracket operator is the bilinear operator defined as

$$[\cdot,\cdot]:\mathfrak{g} imes\mathfrak{g} o\mathfrak{g},\qquad \left[\widetilde{\mathbf{a}},\widetilde{\mathbf{b}}
ight]\mapsto d_b(\widetilde{\mathbf{a}})-d_a(\widetilde{\mathbf{b}})$$

Cross derivatives

$$d_b(\widetilde{\mathbf{a}}) - d_a(\widetilde{\mathbf{b}}) = \left[\widetilde{\mathbf{a}}, \widetilde{\mathbf{b}}\right]$$
$$d_b(\mathbf{a}) - d_a(\mathbf{b}) = \widehat{\mathbf{a}}\mathbf{b} = \mathrm{ad}_{\mathbf{a}}\mathbf{b} \qquad \widehat{\mathbf{a}} = \begin{bmatrix} \widetilde{\mathbf{a}}_{\omega} & \widetilde{\mathbf{a}}_{u} \\ \mathbf{0}_{3\times3} & \widetilde{\mathbf{a}}_{\omega} \end{bmatrix}$$

Definition

The linear operator $\widehat{(\cdot)}$ is the bilinear operator defined as

$$\widehat{(\cdot)}: \mathbb{R}^k \to \mathbb{R}^{k imes k}, \qquad \mathbf{a} \mapsto \widehat{\mathbf{a}} = \mathbf{A}$$

Strain energy, costitutive equations and stiffness matrix

Strain energy

$$\mathcal{V}_{int} = rac{1}{2} \int_{L} \boldsymbol{f}^{T} \boldsymbol{K} \boldsymbol{f} \, \mathrm{d} lpha$$

Costitutive equations

$$\boldsymbol{\sigma}(\alpha) = \mathbf{K}(\alpha) \, \boldsymbol{f}(\alpha)$$

Strain energy, costitutive equations and stiffness matrix (cont'd)

Stiffness matrix

$$oldsymbol{K} = egin{bmatrix} oldsymbol{K}_{uu} & oldsymbol{K}_{u\omega} \ SYM & oldsymbol{K}_{\omega\omega} \end{bmatrix}$$

Initially straight beam + reference curve \equiv neutral axis of the beam (i.e. \mathbf{n}^0 and \mathbf{b}^0 are chosen to be the principal axes of the cross-sections) $\rightarrow \mathbf{K}(\alpha_1)$ is diagonal:

$$\mathbf{K}_{uu} = \operatorname{diag}(EA, GA_2, GA_3)$$

$$\mathbf{K}_{\omega\omega} = \operatorname{diag}(GJ, EI_2, EI_3)$$

Static equilibrium equations

Principle of Virtual Work

 $\delta(\mathcal{V}_{int}) = \delta(\mathcal{V}_{ext})$

Variations

Variations of a Lie group element

$$\delta(\mathbf{R}) = \mathbf{R}\widetilde{\delta \boldsymbol{\theta}}$$
$$\delta(\mathbf{H}) = \mathbf{H}\widetilde{\delta \mathbf{h}}$$

where $\delta \theta \in \mathfrak{so}(3)$ is an arbitrary infinitesimal rotation associated with the axial vector $\delta \theta \in \mathbb{R}^3$ and $\delta \mathbf{h}_u = \mathbf{R}^T \delta \mathbf{u} \in \mathbb{R}^3$ is an arbitrary infinitesimal displacement

Variations of a twist element

$$egin{aligned} \delta(\widetilde{oldsymbol{\eta}}) - (\widetilde{\delta f h})^{\cdot} &= \left[\widetilde{oldsymbol{\eta}}, \widetilde{\delta f h}
ight] \ \delta(oldsymbol{\eta}) - (\delta f h)^{\cdot} &= \widehat{oldsymbol{\eta}} \delta f h = - \widehat{\delta f h} oldsymbol{\eta} \end{aligned}$$

Static equilibrium equations

Principle of Virtual Work

 $\delta(\mathcal{V}_{int}) = \delta(\mathcal{V}_{ext})$

$$\delta(\mathbf{f}) = (\delta \mathbf{h})' + \widehat{\mathbf{f}} \delta \mathbf{h}$$

$$\delta(\mathcal{V}_{int}) = \int_{L} \delta(\mathbf{f})^{T} \boldsymbol{\sigma} \, \mathrm{d}\alpha = \\ = \left[\delta \mathbf{h}^{T} \boldsymbol{\sigma} \right] |_{0}^{L} - \int_{L} \delta \mathbf{h}^{T} (\boldsymbol{\sigma}' - \hat{\mathbf{f}}^{T} \boldsymbol{\sigma}') \, \mathrm{d}\alpha$$

$$\delta(\mathcal{V}_{ext}) = +\delta \mathbf{h}(0)^T \mathbf{g}_{ext}(0) - \delta \mathbf{h}(L)^T \mathbf{g}_{ext}(L) - \int_L \delta \mathbf{h}^T \mathbf{g}_{ext}(\alpha) \, \mathrm{d}\alpha$$

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$$\mathbf{g}_{ext}(\alpha) = \begin{bmatrix} \mathbf{g}_{ext,u}^{\mathsf{T}} & \mathbf{g}_{ext,\omega}^{\mathsf{T}} \end{bmatrix}$$

Static equilibrium equations (cont'd)

Static equilibrium equations

weak form
$$\begin{bmatrix} \delta \mathbf{h}^T (\boldsymbol{\sigma} - \mathbf{g}_{ext}) \end{bmatrix} |_0^L - \int_L \delta \mathbf{h}^T (\boldsymbol{\sigma}' - \hat{\mathbf{f}}^T \boldsymbol{\sigma} - \mathbf{g}_{ext}) d\alpha = 0$$

strong form $\boldsymbol{\sigma}' - \hat{\mathbf{f}}^T \boldsymbol{\sigma} = \mathbf{g}_{ext}$

Velocity field

 $\dot{\boldsymbol{H}}(\alpha) = \boldsymbol{H}(\alpha)\widetilde{\boldsymbol{\eta}}(\alpha)$

$$\begin{split} \widetilde{\boldsymbol{\eta}} &= \begin{bmatrix} \widetilde{\boldsymbol{\omega}} & \boldsymbol{v} \\ \boldsymbol{0}_{1\times 3} & 1 \end{bmatrix} \in \mathfrak{se}(3) \\ \boldsymbol{v} &\in \mathbb{R}^3 \\ \widetilde{\boldsymbol{\omega}} &= \begin{bmatrix} 0 & -\omega_3 & -\omega_2 \\ -\omega_3 & 0 & -\omega_1 \\ -\omega_2 & -\omega_1 & 0 \end{bmatrix} \in \mathfrak{so}(3) \end{split}$$

velocity twist

vector of linear velocities

3) skew-symmetric matrix of angular velocities

Kinetic energy

$$\mathcal{K} = rac{1}{2} \int_{L} \boldsymbol{\eta}^{T} \mathbf{M} \boldsymbol{\eta} \, \mathrm{d} lpha$$

$$\mathbf{M} = \begin{bmatrix} \rho A \mathbf{I}_{3 \times 3} & \mathbf{J}_{I}^{T} \\ \mathbf{J}_{I} & \mathbf{J}_{II} \end{bmatrix}$$

- ρ density
- A cross-section area
- J_1 first moment of inertia of the cross section (computed in the local axes of the arm)
- ${\bf J}_{\it II}\,$ second moment of inertia of the cross section (computed in the local axes of the arm)

Dynamic equilibrium equations

Hamilton's principle

$$\int_{t_0}^{t_1} \left(\delta(\mathcal{K}) - \delta(\mathcal{V}_{int}) + \delta(\mathcal{V}_{ext})
ight) dt = 0 \; .$$

$$\delta(\mathcal{V}_{int}) = \int_{L} \delta(\mathbf{f})^{T} \boldsymbol{\sigma} \, \mathrm{d}\alpha =$$
$$= \left[\delta \mathbf{h}^{T} \boldsymbol{\sigma} \right] |_{0}^{L} - \int_{L} \delta \mathbf{h}^{T} (\boldsymbol{\sigma}' - \hat{\mathbf{f}}^{T} \boldsymbol{\sigma}') \, \mathrm{d}\alpha$$

$$\delta(\mathcal{V}_{ext}) = +\delta \mathbf{h}(0)^{T} \mathbf{g}_{ext}(0) - \delta \mathbf{h}(L)^{T} \mathbf{g}_{ext}(L) - \int_{L} \delta \mathbf{h}^{T} \mathbf{g}_{ext}(\alpha) \, \mathrm{d}\alpha$$

Dynamic equilibrium equations (cont'd)

Hamilton's principle

$$\int_{t_0}^{t_1} \left(\delta(\mathcal{K}) - \delta(\mathcal{V}_{\textit{int}}) + \delta(\mathcal{V}_{\textit{ext}})
ight) dt = 0 \; .$$

$$\delta(oldsymbol{\eta}) = (\dot{\delta \mathbf{h}}) + \widehat{oldsymbol{\eta}} \delta \mathbf{h}$$

$$\int_{t_0}^{t_1} \delta(\mathcal{K}) dt = \int_{t_0}^{t_1} \int_L \delta(\eta)^T \mathbf{M} \eta \, d\alpha dt = \\ = \left[\int_L \delta \mathbf{h}^T \mathbf{M} \eta \, d\alpha \right]_{t_0}^{t_1} - \int_{t_0}^{t_1} \int_L \delta \mathbf{h}^T (\mathbf{M} \dot{\eta} - \hat{\eta}^T \mathbf{M} \eta) \, d\alpha \, dt$$

Dynamic equilibrium equations (cont'd)

Dynamic equilibrium equations

weak form
$$\begin{bmatrix} \delta \mathbf{h}^{T} (\boldsymbol{\sigma} - \mathbf{g}_{ext}) \end{bmatrix} |_{0}^{L} - \int_{L} \delta \mathbf{h}^{T} (-\mathbf{M}\dot{\boldsymbol{\eta}} + \hat{\boldsymbol{\eta}}^{T} \mathbf{M} \boldsymbol{\eta} + \boldsymbol{\sigma}' - \hat{\mathbf{f}}^{T} \boldsymbol{\sigma} + \mathbf{g}_{ext}) d\alpha = 0$$

strong form $\mathbf{M}\dot{\boldsymbol{\eta}} - \hat{\boldsymbol{\eta}}^{T} \mathbf{M} \boldsymbol{\eta} - \boldsymbol{\sigma}' + \mathbf{f}^{T} \boldsymbol{\sigma} = \mathbf{g}_{ext}$
Equations of motion

Kinematic equations $\dot{\mathbf{H}} = \mathbf{H}\tilde{\eta}$
 $\mathbf{H}' = \mathbf{H}\tilde{f}$ Material constitutive law $\boldsymbol{\sigma} = \mathbf{K}f$ Compatibility equations $\eta' - \dot{f} = \hat{\eta}f$ Boundary conditions $\delta h(L) (\mathbf{K}(L)f(L) - \mathbf{g}_{ext}(L)) = \delta h(0) (\mathbf{K}(0)f(0) - \mathbf{g}_{ext}(0))$ Dynamic equilibrium equations $\mathbf{M}\dot{\eta} - \hat{\eta}^T \mathbf{M}\eta - \boldsymbol{\sigma}' + f^T \boldsymbol{\sigma} = \mathbf{g}_{ext}$

Part B: Discrete formulation with piecewise constant deformation

The continuous field of deformation $f(\alpha)$ leads to dynamics described by PDE!

Continuum mechanics	Robotics
$\mathbf{M}\dot{oldsymbol{\eta}}-\widehat{oldsymbol{\eta}}^{T}\mathbf{M}oldsymbol{\eta}-oldsymbol{\sigma}'+oldsymbol{f}^{T}oldsymbol{\sigma}=\mathbf{g}_{ext}$	$M\ddot{\bm{q}}+C\dot{\bm{q}}+\bm{g}=\bm{\tau}$

How to close the gap?







From the continuum formulation ...

 α_{f}

continuum arm: continuous field $f(\alpha)$

.... To the discrete formulation



... To the discrete formulation



... To the discrete formulation



Grazioso, Di Gironimo, Siciliano "A geometrically exact model for soft continuum robots" Soft Robotics, 2019

Kinematics

$$\mathbf{H}'(\alpha) = \mathbf{H}(\alpha)\widetilde{\mathbf{f}}(\alpha)$$

$$\Downarrow$$

\ldots since ${\it f}$ does not depend on lpha

• **f**: constant deformation vector of the discrete element of the arm

Kinematics mapping

$$\boldsymbol{H}(\alpha) = \boldsymbol{H}_{0} \exp_{\boldsymbol{SE}(3)} \left(\alpha \widetilde{\boldsymbol{f}} \right)$$

• H_0 in SE(3): configuration of the arm at $\alpha = 0$

Kinematics

The exponential map on SE(3)

$$\begin{aligned} & \mathsf{xp}_{SE(3)}(\cdot) : \mathbb{R}^6 \to SE(3), \qquad \boldsymbol{f} \mapsto \mathsf{exp}_{SE(3)}(\boldsymbol{f}) \quad (*) \\ & \mathsf{exp}_{SE(3)}(\boldsymbol{f}) = \begin{bmatrix} \mathsf{exp}_{SO(3)}(\boldsymbol{f}_\omega) & \boldsymbol{T}_{SO(3)}^T(\boldsymbol{f}_\omega)\boldsymbol{f}_u \\ \boldsymbol{0}_{1\times 3} & 1 \end{bmatrix} \end{aligned}$$

•
$$\exp_{SO(3)}(\boldsymbol{f}_{\omega}) = \boldsymbol{I}_{3\times3} + \alpha(\boldsymbol{f}_{\omega})\tilde{\boldsymbol{f}}_{\omega} + \frac{\beta(\boldsymbol{f}_{\omega})}{2}\tilde{\boldsymbol{f}}_{\omega}^{2}$$
: Rodrigues' formula
• $\boldsymbol{T}_{SO(3)}(\boldsymbol{f}_{\omega}) = \boldsymbol{I}_{3\times3} - \frac{\beta(\boldsymbol{f}_{\omega})}{2}\tilde{\boldsymbol{f}}_{\omega} + \frac{1-\alpha(\boldsymbol{f}_{\omega})}{\|\boldsymbol{f}_{\omega}\|^{2}}\tilde{\boldsymbol{f}}_{\omega}^{2}$: Tangent operator
 $\alpha(\boldsymbol{f}_{\omega}) = \frac{\sin(\|\boldsymbol{f}_{\omega}\|)}{\|\boldsymbol{f}_{\omega}\|} \qquad \beta(\boldsymbol{f}_{\omega}) = 2\frac{1-\cos(\|\boldsymbol{f}_{\omega}\|)}{\|\boldsymbol{f}_{\omega}\|^{2}}$

(*) The formal definition of exponential map uses the Lie algebra $\mathfrak{se}(3)$ instead of \mathbb{R}^6 . However, due to the isomorphism between Lie algebra $\mathfrak{se}(3)$ and \mathbb{R}^6 , Hence, with a slight abuse of notation, we use \mathbb{R}^6 instead of $\mathfrak{se}(3)$

Exponential map

Definition

The exponential map projects an element of the Lie algebra into an element of the Lie group

$$exp: \mathfrak{g}
ightarrow G, \quad \widetilde{\mathbf{a}} \mapsto exp(\widetilde{\mathbf{a}})$$

and it is given by

$$exp(\widetilde{\mathbf{a}}) = \sum_{i=0}^{\infty} \frac{\widetilde{\mathbf{a}}^i}{i!}$$

Exponential map (cont'd)



Kinematics – multiple elements

$$\mathbf{H}'(\alpha) = \mathbf{H}(\alpha) \,\widetilde{\mathbf{f}}(\alpha)$$

 $\downarrow \downarrow$

- $\alpha \in [0, L_n] = [0, L_1), (L_1, L_2), \dots, (L_{n-1}, L_n]$ L_n : total length of the arm
- **f**_i: constant deformation vector of each discrete element of the arm

Product of exponentials (PoE)

$$\boldsymbol{H}(\alpha) = \boldsymbol{H}_0 \prod_{i=1}^n \exp_{SE(3)} \left((\min(L_i, \alpha_i) - L_{i-1}) \widetilde{\boldsymbol{f}}_i \right)$$

• H_0 in SE(3): configuration of the arm at $\alpha = 0$

 \ldots since $m{f}$ does not depend on lpha

Inverse kinematics - one element

Mapping

$$k_{I}(\cdot): \quad \mathbf{H}_{0}, \mathbf{H}_{L} \in SE(3) \mapsto \mathbf{f} = \log_{SE(3)} \left(\mathbf{H}_{0}^{-1}\mathbf{H}_{L}\right) \in \mathbb{R}^{6}$$

- $\mathbf{H}_0, \mathbf{H}_L$: the configuration of the arm at $\alpha = 0, \alpha = L$
- $\log_{SE(3)}(\cdot)$: the logarithmic map on SE(3)

Inverse kinematics (cont'd)

The logarithmic map on
$$SE(3)$$

$$\log_{SE(3)}(\cdot) : SE(3) \to \mathbb{R}^6, \qquad \mathbf{H} \mapsto \log_{SE(3)}(\mathbf{H}) \quad (*)$$

$$\log_{SE(3)}(\mathbf{H}) = \begin{bmatrix} \log_{SO(3)}(\mathbf{R}) & \mathbf{T}_{SO(3)}^{-T}(\mathbf{f}_{\omega})\mathbf{u} \\ \mathbf{0}_{1\times 3} & 1 \end{bmatrix}$$

•
$$\log_{SO(3)}(\mathbf{R}) = \frac{\theta}{2\sin\theta}(\mathbf{R} - \mathbf{R}^{T})$$
, with $\theta = \operatorname{acos}\left(\frac{1}{2}(\operatorname{trace}(\mathbf{R}) - 1)\right)$, $\theta < \pi$
• $\mathbf{T}_{SO(3)}^{-1}(\mathbf{f}_{\omega}) = \mathbf{I}_{3\times 3} + \frac{1}{2}\widetilde{\mathbf{f}}_{\omega} + \frac{1 - \gamma(\mathbf{f}_{\omega})}{\|\mathbf{f}_{\omega}\|^{2}}\widetilde{\mathbf{f}}_{\omega}^{2}$: Inverse of the tangent operator
 $\Psi(\mathbf{f}_{\omega}) = \frac{\|\mathbf{f}_{\omega}\|}{2}\operatorname{cot}\left(\frac{\|\mathbf{f}_{\omega}\|}{2}\right)$

(*) The formal definition of logarithmic map uses the Lie algebra $\mathfrak{se}(3)$ instead of \mathbb{R}^6 . As before, since the isomorphism $\mathfrak{se}(3) \simeq \mathbb{R}^6$ holds, with a slight abuse of notation, we use \mathbb{R}^6 .

Logarithmic map

Definition

The logarithmic map projects an element of the Lie group into an element of the Lie algebra

$$\mathit{log}: \mathit{G}
ightarrow \mathfrak{g}, \quad \mathit{q} \mapsto \mathit{log}(\mathit{q}) = \widetilde{\mathsf{a}}$$

and it is given by

$$\mathit{log}(q) = \sum_{i=0}^\infty rac{(e-q)^i}{i}$$

Tangent map

Definition

$$\mathsf{T}: \mathbb{R}^k o \mathbb{R}^k, \quad \mathsf{u} \mapsto \mathsf{T}(\mathsf{u}) d_{\mathsf{a}}(\mathsf{u}) = \mathsf{a}$$

with

$$\mathbf{T}(\mathbf{u}) = \sum_{i=0}^{\infty} (-1)^i \frac{\widehat{\mathbf{u}}^i}{(i+1)!}$$

Inverse of the tangent map

Definition

$$\mathsf{T}^{-1}:\mathbb{R}^k o\mathbb{R}^k,\quad \mathsf{u},\mathsf{a}\mapsto\mathsf{T}^{-1}(\mathsf{u})\mathsf{a}=d_{\mathsf{a}}(\mathsf{u})$$

with

$$\mathbf{T}^{-1}(\mathbf{u}) = \sum_{i=0}^{\infty} (-1)^i B_i \frac{\widehat{\mathbf{u}}^i}{(i)!}$$

where B_i is the Bernoulli number of the first kind.

The exponential map on SO(3)

Exponential map
$$\exp_{SO(3)}(\mathbf{h}_{\omega}) = \mathbf{I}_{3\times3} + \alpha(\mathbf{h}_{\omega})\widetilde{\mathbf{h}}_{\omega} + \frac{\beta(\mathbf{h}_{\omega})}{2}\widetilde{\mathbf{h}}_{\omega}^{2}$$
Logarithmic map $\log_{SO(3)}(\mathbf{R}) = \frac{\theta}{2\sin\theta}(\mathbf{R} - \mathbf{R}^{T})$ Tangent operator $\mathbf{T}_{SO(3)}(\mathbf{h}_{\omega}) = \mathbf{I}_{3\times3} - \frac{\beta(\mathbf{h}_{\omega})}{2}\widetilde{\mathbf{h}}_{\omega} + \frac{1-\alpha(\mathbf{h}_{\omega})}{\|\mathbf{h}_{\omega}\|^{2}}\widetilde{\mathbf{h}}_{\omega}^{2}$ Inverse of the tangent operator $\mathbf{T}_{SO(3)}^{-1}(\mathbf{h}_{\omega}) = \mathbf{I}_{3\times3} + \frac{1}{2}\widetilde{\mathbf{h}}_{\omega} + \frac{1-\gamma(\mathbf{h}_{\omega})}{\|\mathbf{h}_{\omega}\|^{2}}\widetilde{\mathbf{h}}_{\omega}^{2}$

$$\begin{aligned} \alpha(\mathbf{h}_{\omega}) &= \frac{\sin(\|\mathbf{h}_{\omega}\|)}{\|\mathbf{h}_{\omega}\|} \quad \beta(\mathbf{h}_{\omega}) = 2\frac{1 - \cos(\|\mathbf{h}_{\omega}\|)}{\|\mathbf{h}_{\omega}\|^{2}} \quad \gamma(\mathbf{h}_{\omega}) = \frac{\|\mathbf{h}_{\omega}\|}{2} \cot\left(\frac{\|\mathbf{h}_{\omega}\|}{2}\right) \\ \theta &= \arccos\left(\frac{1}{2}(\operatorname{trace}(\mathbf{R}) - 1\right), \quad \theta < \pi \end{aligned}$$

The exponential map on SE(3)

Exponential map	$\exp_{SE(3)}(\mathbf{h}) = \begin{bmatrix} \exp_{SO(3)}(\mathbf{h}_{\omega}) & \mathbf{T}_{SO(3)}^{T}(\mathbf{h}_{\omega})\mathbf{h}_{u} \\ 0_{1\times 3} & 1 \end{bmatrix}$
Logarithmic map	$log_{SE(3)}(H) = egin{bmatrix} \widetilde{h}_{\omega} & T_{SO(3)}^{-\mathcal{T}}(h_{\omega})h_{u} \ 0_{1 imes 3} & 0 \end{bmatrix}$
Tangent operator	$T_{\mathcal{SE}(3)}(h) = egin{bmatrix} T_{\mathcal{SO}(3)}(h_{\omega}) & T_{u\omega+}(h_{u},h_{\omega}) \ 0_{3 imes 3} & T_{\mathcal{SO}(3)}(h_{\omega}) \end{bmatrix}$
Inverse of the tangent operator	$\mathbf{T}_{SE(3)}^{-1}(\mathbf{h}) = egin{bmatrix} \mathbf{T}_{SO(3)}^{-1}(\mathbf{h}_{\omega}) & \mathbf{T}_{u\omega-}(\mathbf{h}_{u},\mathbf{h}_{\omega}) \ 0_{3 imes 3} & \mathbf{T}_{SO(3)}^{-1}(\mathbf{h}_{\omega}) \end{bmatrix}$

The exponential map on SE(3) (cont'd)

$$\begin{split} \mathbf{T}_{u\omega+}(\mathbf{h}_{\omega},\mathbf{h}_{u}) &= \frac{-\beta}{2}\widetilde{\mathbf{h}}_{\omega} + \frac{1-\alpha}{\|\mathbf{h}_{\omega}\|^{2}}[\mathbf{h}_{\omega},\mathbf{h}_{u}] + \frac{\mathbf{h}_{u}^{T}\mathbf{h}_{\omega}}{\|\mathbf{h}_{u}\|^{2}}\left((\beta-\alpha)\widetilde{\mathbf{h}}_{u} + (\frac{\beta}{2} - \frac{3(1-\alpha)}{\|\mathbf{h}_{u}\|^{2}})\widetilde{\mathbf{h}}_{u}^{2}\right)\\ \mathbf{T}_{u\omega-}(\mathbf{h}_{\omega},\mathbf{h}_{u}) &= \frac{1}{2}\widetilde{\mathbf{h}}_{\omega} + \frac{1-\gamma}{\|\mathbf{h}_{\omega}\|^{2}}[\mathbf{h}_{\omega},\mathbf{h}_{u}] + \frac{\mathbf{h}_{u}^{T}\mathbf{h}_{\omega}}{\|\mathbf{h}_{u}\|^{4}}\left((\frac{1}{\beta}+\gamma-2)\widetilde{\mathbf{h}}_{u}^{2}\right) \end{split}$$

Geometric interpretation of the interpolated reference curve

$$\mathbf{H}(\alpha) = \mathbf{H}_{A}\mathbf{H}_{A0} \exp_{SE(3)} \left(\frac{\alpha}{L}\widetilde{\mathbf{d}}\right)$$

$$\downarrow$$

$$\mathbf{u}'(\alpha) = \mathbf{R}(\alpha)\frac{\widetilde{\mathbf{d}}_{u}}{L} \text{ and } \mathbf{R}'(\alpha) = \mathbf{R}(\alpha)\frac{\widetilde{\mathbf{d}}_{u}}{L}$$

Local triad

$$\mathbf{t}(\alpha) = \frac{\mathbf{u}'(\alpha)}{\|\mathbf{u}(\alpha)\|} = \mathbf{R}(\alpha)\frac{\mathbf{d}_u}{L}$$
$$\mathbf{n}(\alpha) = \frac{1}{\kappa}\mathbf{t}'(\alpha) = \mathbf{R}(\alpha)\frac{\widetilde{\mathbf{d}}_\omega \mathbf{d}_u}{\|\widetilde{\mathbf{d}}_\omega \mathbf{d}_u\|}$$
$$\mathbf{b}(\alpha) = \frac{1}{\|\mathbf{d}_u\|\|\widetilde{\mathbf{d}}_\omega \mathbf{d}_u\|}\widetilde{\mathbf{R}(\alpha)\mathbf{d}_u}\mathbf{R}(\alpha)\mathbf{d}_\omega\mathbf{d}_u = \mathbf{R}(\alpha)\frac{\widetilde{\mathbf{d}}_u\widetilde{\mathbf{d}}_\omega\mathbf{d}_u}{\|\mathbf{d}_u\|\|\widetilde{\mathbf{d}}_\omega\mathbf{d}_u\|}$$

Geometric interpretation of the interpolated reference curve (cont'd)

Curvature and torsion of the curve

$$\kappa = \frac{\|\widetilde{\mathbf{d}}_{\omega}\mathbf{d}_{u}\|}{L^{2}} = \frac{\|\mathbf{d}_{\omega}\|\sin(\mathbf{d}_{\omega}, \mathbf{d}_{u})}{L} = \text{cost}$$
$$\tau = \frac{(\mathbf{d}_{\omega}^{\mathsf{T}}\mathbf{d}_{u})}{L^{2}} = \frac{\|\mathbf{d}_{\omega}\|\cos(\mathbf{d}_{\omega}, \mathbf{d}_{u})}{L} = \text{cost}$$
$$\kappa_{g} = \sqrt{\kappa^{2} + \tau^{2}} = \frac{\|\mathbf{d}_{\omega}\|}{L} = \frac{\|\mathbf{d}_{\omega}\|}{\|\mathbf{d}_{u}\|} = \text{cost}$$

Geometric interpretation of the interpolated reference curve (cont'd)



Curvature and torsion of the curve

$$\kappa = \frac{\|\widetilde{\mathbf{d}}_{\omega}\mathbf{d}_{u}\|}{L^{2}} = \frac{\|\mathbf{d}_{\omega}\|\sin(\mathbf{d}_{\omega},\mathbf{d}_{u})}{L} = \text{cost}$$
$$\tau = \frac{(\mathbf{d}_{\omega}^{\mathsf{T}}\mathbf{d}_{u})}{L^{2}} = \frac{\|\mathbf{d}_{\omega}\|\cos(\mathbf{d}_{\omega},\mathbf{d}_{u})}{L} = \text{cost}$$

Geometric interpretation of the interpolated reference curve (cont'd)



Physical meaning: Helix!

Example

Computing the shape of a soft arm with one element (n = 1) with internal deformations f_u = [100]^T; f_ω = [τ0κ]^T (no axial and shear deformations; bending about z; torsion about x)

$$\boldsymbol{H}(\alpha) = \begin{bmatrix} 1 - (1 - \cos(\alpha\kappa_g))\frac{\kappa^2}{\kappa_g^2} & -\sin(\alpha\kappa_g)\frac{\kappa}{\kappa_g} & (1 - \cos(\alpha\kappa_g))\frac{\kappa\tau}{\kappa_g^2} & \alpha + (\sin(\alpha\kappa_g) - \alpha\kappa_g)\frac{\kappa^2}{\kappa_g^3} \\ \sin(\alpha\kappa_g)\frac{\kappa}{\kappa_g} & \cos(\alpha\kappa_g) & -\sin(\alpha\kappa_g)\frac{\tau}{\kappa_g} & (1 - \cos(\alpha\kappa_g))\frac{\kappa}{\kappa_g^2} \\ (1 - \cos(\alpha\kappa_g))\frac{\kappa\tau}{\kappa_g^2} & \sin(\alpha\kappa_g)\frac{\tau}{\kappa_g} & 1 - (1 - \cos(\alpha\kappa_g))\frac{\tau^2}{\kappa_g^2} & (\alpha\kappa_g - \sin(\alpha\kappa_g))\frac{\kappa\tau}{\kappa_g^3} \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

∜

• $\kappa_g = \sqrt{\kappa^2 + \tau^2}$: Gaussian curvature of the arm

Grazioso, Di Gironimo, Siciliano "From differential geometry of curves to helical kinematics of continuum robots using exponential mapping" ARK, 2018

Example (cont'd)



(a) $\tau = 3 \text{ m}^{-1}$; $\kappa = 0 : \pi/2 : 2\pi \text{ m}^{-1}$. (b) $\kappa = 3 \text{ m}^{-1}$; $\tau = 0 : \pi/2 : 2\pi \text{ m}^{-1}$

Figure: Whole arm screw motion of a manipulator with constant curvature and torsion. L = 1 m.

Grazioso, Di Gironimo, Siciliano "From differential geometry of curves to helical kinematics of continuum robots using exponential mapping" ARK, 2018

Differential kinematics

• Mapping between the velocities along the arm and the time derivatives of the states of the manipulator, i.e. the internal deformations

Differential kinematics

 $\boldsymbol{\eta}(\alpha) = \boldsymbol{J}(\alpha, \boldsymbol{d}) \boldsymbol{f}$

Soft Geometric Jacobian

$$J(\alpha, \boldsymbol{d}) = \operatorname{Ad}_{\exp_{SE(3)}\left(-\frac{\alpha}{L}\boldsymbol{d}\right)} \mathbf{H}_{A0}^{-1} \mathbf{T}_{SE(3)}^{-1} (-\boldsymbol{d}) \operatorname{Ad}_{\mathbf{H}_{A0}^{-1}} \cdots \\ \cdots \left(\mathbf{T}_{SE(3)}^{-1} (-\boldsymbol{d}) \operatorname{Ad}_{\mathbf{H}_{A0}^{-1}} \right)^{2} + (\mathbf{T}_{SE(3)}^{-1} (\boldsymbol{d}) \operatorname{Ad}_{\mathbf{H}_{B0}^{-1}} \right)^{2})^{-1} \right) \cdots \\ \cdots + \frac{\alpha}{L} \mathbf{T}_{SE(3)} \left(\frac{\alpha}{L} \boldsymbol{d} \right)$$

Adjoint representation

Definition

The adjoint representation of a Lie algebra element is defined as

$$\mathsf{A}d_q:\mathfrak{g}
ightarrow\mathfrak{g},\qquad \widetilde{\mathbf{a}}\mapsto q\widetilde{\mathbf{a}}q^-$$

Adjoint representation of a $\mathfrak{se}(3)$ element

$$\begin{aligned} \mathsf{Ad}_{\mathsf{H}}(\widetilde{\mathsf{a}}) &= \mathsf{H}\widetilde{\mathsf{a}}\mathsf{H}^{-1} \\ \mathsf{Ad}_{\mathsf{H}}(\mathsf{a}) &= \begin{bmatrix} \mathsf{R} & \widetilde{\mathsf{u}}\mathsf{R} \\ \mathbf{0}_{3\times 3} & \mathsf{R} \end{bmatrix} \mathsf{a} \end{aligned}$$

Statics

Variation of internal energy (continuum)

$$\delta(\mathcal{V}_{int}) = \int_{L} \delta(\boldsymbol{f}(\alpha))^{T} \boldsymbol{\mathsf{K}}(\alpha) \boldsymbol{f}(\alpha) \, \mathrm{d}\alpha$$

• f = cost for the finite element

Variation of internal energy (discrete)

∜

$$\delta(\mathcal{V}_{int}) = \delta(\mathbf{f})^T \mathbf{K}_L \mathbf{f}, \qquad \mathbf{K}_L = \int_L \mathbf{K}(\alpha) \, \mathrm{d}\alpha; \quad \mathbf{K}_L \mathbf{f} = \mathbf{T}$$

Statics

Variation of external energy (continuum)

$$\delta(\mathcal{V}_{ext}) = +\delta \boldsymbol{h}(0)^{T} \mathbf{g}_{ext}(0) - \delta \boldsymbol{h}(L)^{T} \mathbf{g}_{ext}(L) - \int_{L} \delta \boldsymbol{h}^{T} \mathbf{g}_{ext} \, \mathrm{d}\alpha$$

•
$$\boldsymbol{f} = \text{cost}$$
 for the finite element
• $\delta \boldsymbol{h}(\alpha) = \boldsymbol{J}(\alpha, \boldsymbol{d})\delta(\boldsymbol{f})$

Variation of external energy (discrete)

∜

$$\delta(\mathcal{V}_{ext}) = -\delta(\mathbf{f})^T \mathbf{J}_L^T(\mathbf{d}) \mathbf{F}, \qquad \mathbf{F} = \int_L \mathbf{g}_{ext} \, \mathrm{d}\alpha; \quad \mathbf{J}_L(\mathbf{d}) = \int_L \mathbf{J}(\alpha, \mathbf{d}) \, \mathrm{d}\alpha$$

Statics



Dynamics

Weak form of dynamic equilibrium equations (continuum)

$$\begin{bmatrix} \delta \mathbf{h}^{T} (\boldsymbol{\sigma} - \mathbf{g}_{ext}) \end{bmatrix} |_{0}^{L} - \int_{L} \delta \mathbf{h}^{T} (-\mathbf{M} \dot{\boldsymbol{\eta}} + \hat{\boldsymbol{\eta}}^{T} \mathbf{M} \boldsymbol{\eta} + \boldsymbol{\sigma}' - \hat{\mathbf{f}}^{T} \boldsymbol{\sigma} + \mathbf{g}_{ext}) \, \mathrm{d}\alpha = \mathbf{0}$$

•
$$\eta(\alpha) = J(\alpha, d)\dot{f}$$

•
$$\dot{\boldsymbol{\eta}}(\alpha) = \boldsymbol{J}(\alpha, \mathbf{d})\ddot{\boldsymbol{f}} + \dot{\boldsymbol{J}}(\alpha, \mathbf{d})\dot{\boldsymbol{f}}$$

•
$$\delta \boldsymbol{h}(\alpha) = \boldsymbol{J}(\alpha, \boldsymbol{d})\delta(\boldsymbol{f})$$

Weak form of dynamic equilibrium equations (discrete)

$$\delta(\mathbf{f})^T \int_L \mathbf{J}^T (\mathbf{M}(\mathbf{J}\ddot{\mathbf{f}} + \dot{\mathbf{J}}\dot{\mathbf{f}}) - \widehat{\mathbf{J}}\dot{\mathbf{f}}^T \mathbf{M}\mathbf{J}\dot{\mathbf{f}} + \widehat{\boldsymbol{\epsilon}}^T \mathbf{K}\mathbf{f} - \mathbf{g}_{ext}) d\alpha = 0$$

Dynamics

- $\int_{L} \mathbf{J}^{T} \mathbf{M} \mathbf{J} d\alpha = \mathbb{M}$, the 6 × 6 discretized mass matrix.
- $\int_{L} \mathbf{J}^{T} \mathbf{M} \dot{\mathbf{J}} d\alpha = \mathbb{C}_{1}$, the 6 × 6 velocity matrix which contributes only if $\dot{\mathbf{e}}$ does not vanish, i.e., only when the deformation of the arm changes in time.
- $\int_{L} \mathbf{J}^{T} \widehat{\mathbf{J}\mathbf{f}}^{T} \mathbf{M} \mathbf{J} d\alpha = \mathbb{C}_{2}$, the 6 × 6 velocity matrix related to gyroscopic effects, contributes also in the case of a rigid body motion of the soft arm.
- $\int_{L} \boldsymbol{J}^{T} \boldsymbol{\hat{f}}^{T} \boldsymbol{\mathsf{K}} \, \mathrm{d}\alpha = \mathbb{K}$, the 6 × 6 discretized stiffness matrix.
- $\int_{L} \mathbf{J}^{T} \mathbf{g}_{ext} d\alpha = \mathbb{F}$, the 6 × 1 vector of generalized applied forces. It also includes actuation loads and gravity field.

Dynamic model

$$\mathbb{M}(\alpha, \boldsymbol{f})\ddot{\boldsymbol{f}} + (\mathbb{C}_1\left(\alpha, \boldsymbol{f}, \dot{\boldsymbol{f}}\right) - \mathbb{C}_2(\alpha, \boldsymbol{f}, \dot{\boldsymbol{f}})\right)\dot{\boldsymbol{f}} - \mathbb{K}\boldsymbol{f} = \mathbb{F}$$
The Princeton Experiment



- θ: 15, 30, 45, 60, 75, 90 degree
- f: 4.448, 8.896, 13,344, 17.792 N

The Princeton Experiment



The Princeton Experiment



Part C:

Piecewise constant curvature as special case of the deformation space formulation

Looking for simplified models!

- Under reference external forces
- Using a particular actuation source (tendons / pneumatic actuation)
- We look for experimental prototypes and we see that they exhibit some relevant shape configuration



Figure: Cantilever soft arm subject to a torque au_y at its free end

Grazioso, Di Gironimo, Siciliano "Analytic solutions for the static equilibrium configurations of externally loaded cantilever soft robotic arms" *ROBOSOFT*, 2018

Equations

Kinematic equations

Material constitutive law

Boundary conditions

Statics

 $\mathbf{H}'(\alpha) = \mathbf{H}(\alpha)(\mathbf{f})$ $\boldsymbol{\sigma}(\alpha) = \mathbf{K}(\alpha)\mathbf{f}(\alpha)$ $\delta \mathbf{h}(L) \left(\mathbf{K}(L)\mathbf{f}(L) - \mathbf{g}_{ext}(L)\right) - \delta \mathbf{h}(0) \left(\mathbf{K}(0)\mathbf{f}(0) - \mathbf{g}_{ext}(0)\right) = 0$ $\boldsymbol{\sigma}'(\alpha) - \hat{\mathbf{f}}^{T}(\alpha)\boldsymbol{\sigma}(\alpha) = \mathbf{g}_{ext}(\alpha)$

Boundary conditions

$$\boldsymbol{\sigma}(L) = \boldsymbol{K}(L)\boldsymbol{f}(L) = \boldsymbol{g}_{ext}(L)$$

Deformation field

Since the stiffness matrix is constant over the arm, the equilibrium equations in the static configuration become

$$oldsymbol{K}oldsymbol{f}'-oldsymbol{\widehat{f}^0}^Toldsymbol{K}oldsymbol{f}=oldsymbol{0}_{6 imes 1}$$

In this case, the solution for the deformation field can be expressed in closed form and it is given by

$$\boldsymbol{f}(\alpha) = \boldsymbol{K}^{-1} \boldsymbol{F}(\alpha) \boldsymbol{K} \boldsymbol{f}_0$$

where f_0 , the deformation at $\alpha = 0$, is a constant of integration and

$$\boldsymbol{F}(\alpha) = \begin{bmatrix} \boldsymbol{L}^{T}(\alpha) & \boldsymbol{0}_{3\times3} \\ \left(\boldsymbol{T}_{SO(3)}(\alpha \boldsymbol{f}_{\omega}^{0}) \alpha \boldsymbol{f}_{u}^{0}\right)^{\widetilde{}} \boldsymbol{L}^{T}(\alpha) & \boldsymbol{L}^{T}(\alpha) \end{bmatrix}$$

with $\boldsymbol{L}(\alpha) = \exp_{SO(3)} \left(\alpha \boldsymbol{f}_{\omega}^{0} \right).$

Deformation field

The boundary conditions become

$$\boldsymbol{\sigma}(L) = \boldsymbol{K}\boldsymbol{K}^{-1}\boldsymbol{F}(L)\boldsymbol{K}\boldsymbol{f}_0 = \boldsymbol{g}_{ext}(L)$$

such that the constant of integration f_0 is given by

$$\boldsymbol{f}_0 = \boldsymbol{K}^{-1}(\boldsymbol{F}(L))^{-1}\boldsymbol{g}_{ext}(L)$$

Therefore, the solution for the deformation field reads

$$\boldsymbol{f}(\alpha) = \boldsymbol{K}^{-1} \boldsymbol{F}(\alpha) (\boldsymbol{F}(L))^{-1} \boldsymbol{g}_{ext}(L)$$

Deformation field

In the special cases of pure bending/torsion solicitations, the external forces are given by

$$oldsymbol{g}_{ext,u}(L) = oldsymbol{0}_{3 imes 1} \ oldsymbol{g}_{ext,\omega}(L) = au oldsymbol{a}$$

where $\tau \in \mathbb{R}$ and $\boldsymbol{a} \in \mathbb{R}^3$ is an arbitrary vector. For an initially straight arm, we have $\boldsymbol{F}(\alpha)(\boldsymbol{F}(L))^{-1} = \boldsymbol{I}_{6\times 6}$. Hence, the deformation field becomes

$$\boldsymbol{f} = \boldsymbol{K}^{-1} \begin{bmatrix} \boldsymbol{0}_{3 imes 1} \\ au \, \boldsymbol{a} \end{bmatrix}$$

Deformation field

Thus, it results that f is constant along the continuum arm. The solution reads

$$egin{bmatrix} m{\gamma} \ m{\kappa} \end{bmatrix} = egin{bmatrix} m{0}_{3 imes 1} \ m{\kappa}_{\omega\omega}^{-1}(aum{a}) \end{bmatrix}$$

where $K_{\omega\omega} = \text{diag}(GJ, EI_y, EI_z)$ contains the torsional and bending stiffnesses of the cross section.

Position field

The position and orientation fields are obtained by solving the kinematic equations. Since the deformation field obtained above involves constant deformations, the kinematic equations can be integrated analitically and the solution for the SE(3) field is given by

$$\boldsymbol{H}(\alpha) = \boldsymbol{H}_0 \exp_{\boldsymbol{SE}(3)}(\alpha \boldsymbol{f})$$

where $H_0 = \mathcal{H}(R_0, u_0)$ is a constant of integration and $\exp_{SE(3)}(\cdot)$ is the exponential map on SE(3). Explicitly, it reads

$$u(\alpha) = u_0 + R_0 T_{SO(3)}^T (\alpha(f_{\omega}^0 + \kappa)) \alpha(f_{u}^0 + \gamma)$$

$$R(\alpha) = R_0 \exp_{SO(3)}(\alpha(f_{\omega}^0 + \kappa))$$

Since we are considering initially straight arm, we have that $\boldsymbol{u}_0 = \boldsymbol{0}_{3 \times 1}$, $\boldsymbol{R}_0 = \boldsymbol{I}_{3 \times 3}$ and $\boldsymbol{f}_u^0 = [1 \ 0 \ 0]^T$, $\boldsymbol{f}_\omega^0 = \boldsymbol{0}_{3 \times 1}$.

Position field

In the case of the figure above, $\mathbf{g}_{ext,\omega}(L) = \tau \begin{bmatrix} 0 & 1 & 0 \end{bmatrix}^T$. According to the deformation field solution, the deformations are given by

$$oldsymbol{\gamma} = egin{bmatrix} 0 & 0 & 0 \end{bmatrix}^{\mathcal{T}} \ oldsymbol{\kappa} = egin{bmatrix} 0 & \kappa_y & 0 \end{bmatrix}^{\mathcal{T}}$$

where $\kappa_y = \tau_y/(EI_y)$. Indeed, according to the SE(3) field solution, the position and rotation fields are given by

$$\boldsymbol{u}(\alpha) = \begin{bmatrix} \frac{1}{\kappa_{y}} \sin(\alpha \kappa_{y}) \\ 0 \\ -\frac{1}{\kappa_{y}} (1 - \cos(\alpha \kappa_{y})) \end{bmatrix}$$
$$\boldsymbol{R}(\alpha) = \begin{bmatrix} \cos(\alpha \kappa_{y}) & 0 & \sin(\alpha \kappa_{y}) \\ 0 & 1 & 0 \\ -\sin(\alpha \kappa_{y}) & 0 & \cos(\alpha \kappa_{y}) \end{bmatrix}$$

Constant moment applied at the tip of the robotic arm (cont'd) By considering $El_y = 1 \text{ Nm}^2$ and L = 1 m, the soft arm's steady-state shape, for bending tip loads $\tau = 1, 2, 3, 4, 5 \text{ Nm}$, is:



If a constant moment is applied at the tip of a robotic arm \rightarrow the mechanics predict a constant curvature result !

Tendon-driven planar soft continuum arm



- $\bullet\,$ Tendon modeled as an extensible string which can not support internal moments or shear forces, but only tension ${\cal T}$
- $\bullet\,$ Frictionless interaction between the tendon and the channel $\rightarrow\,$ T is constant
- Location of the tendons is not varying under the applied loads
- Constant cross-section properties (mass and stiffness properties)
 - \rightarrow Equivalence between full coupled model (a) and point moment model (b).

Soft continuum arm with constant curvature

•
$$\mathbf{f}_{u} = [1 \ 0 \ 0]^{T}$$
 \xrightarrow{FKINE}
• $\mathbf{f}_{\omega} = [0 \ 0 \ \kappa]^{T}$ \xrightarrow{IKINE} $\mathbf{H}(\alpha) = \begin{bmatrix} \cos(\alpha \kappa) & -\sin(\alpha \kappa) & 0 & \frac{1}{\kappa}\sin(\alpha \kappa) \\ \sin(\alpha \kappa) & \cos(\alpha \kappa) & 0 & \frac{1}{\kappa}(1 - \cos(\alpha \kappa)) \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$

Soft continuum arm with constant curvature



Figure: Whole arm shape configuration under a desired circular end point trajectory. L = 1 m; $\kappa = \pi/10 : \pi \text{ m}^{-1}$.

An alternative approach for kinematic shape recostruction

Kinematics using the exponential map

$$\mathbf{f} \in \mathbb{R}^{6} \mapsto \mathbf{H}(\alpha) = \exp_{SE(3)}(\alpha \mathbf{f}) \in SE(3)$$

•
$$\exp_{SE(3)}(\mathbf{f}) =$$

 $\begin{bmatrix} \exp_{SO(3)}(\mathbf{f}_{\omega}) & \mathbf{T}_{SO(3)}^{T}(\mathbf{f}_{\omega})\mathbf{f}_{u} \\ \mathbf{0}_{1\times 3} & 1 \end{bmatrix}$

•
$$\exp_{SO(3)}(\mathbf{f}_{\omega}) = \mathbf{I}_{3\times 3} + \alpha(\mathbf{f}_{\omega})\widetilde{\mathbf{f}}_{\omega} + \frac{\beta(\mathbf{f}_{\omega})}{2}\widetilde{\mathbf{f}}_{\omega}^{2}$$

•
$$\mathbf{T}_{SO(3)}(\mathbf{f}_{\omega}) = \mathbf{I}_{3\times 3} - \frac{\beta(\mathbf{f}_{\omega})}{2} \mathbf{\tilde{f}}_{\omega} + \frac{1-\alpha(\mathbf{f}_{\omega})}{\|\mathbf{f}_{\omega}\|^2} \mathbf{\tilde{f}}_{\omega}^2$$

 $\alpha(\mathbf{f}_{\omega}) = \frac{\sin(\|\mathbf{f}_{\omega}\|)}{\|\mathbf{f}_{\omega}\|} \qquad \beta(\mathbf{f}_{\omega}) = 2\frac{1-\cos(\|\mathbf{f}_{\omega}\|)}{\|\mathbf{f}_{\omega}\|^2}$

Kinematics using the exponential map (cont'd)

$$\mathbf{f} \in \mathbb{R}^{6} \mapsto \mathbf{H}(lpha) = \exp_{SE(3)}\left(lpha \mathbf{f}\right) \in SE(3)$$

•
$$\exp_{SE(3)}(\mathbf{f}) = \begin{bmatrix} \exp_{SO(3)}(\mathbf{f}_{\omega}) & \mathbf{T}_{SO(3)}^{T}(\mathbf{f}_{\omega})\mathbf{f}_{u} \\ \mathbf{0}_{1 \times 3} & 1 \end{bmatrix}$$

•
$$\exp_{SO(3)}(\mathbf{f}_{\omega}) = \mathbf{I}_{3\times 3} + \alpha(\mathbf{f}_{\omega})\widetilde{\mathbf{f}}_{\omega} + \frac{\beta(\mathbf{f}_{\omega})}{2}\widetilde{\mathbf{f}}_{\omega}^2$$

•
$$\mathbf{T}_{SO(3)}(\mathbf{f}_{\omega}) = \mathbf{I}_{3 \times 3} - \frac{\beta(\mathbf{f}_{\omega})}{2} \widetilde{\mathbf{f}}_{\omega} + \frac{1 - \alpha(\mathbf{f}_{\omega})}{\|\mathbf{f}_{\omega}\|^2} \widetilde{\mathbf{f}}_{\omega}^2$$

 $\alpha(\mathbf{f}_{\omega}) = \frac{\sin(\|\mathbf{f}_{\omega}\|)}{\|\mathbf{f}_{\omega}\|} \qquad \beta(\mathbf{f}_{\omega}) = 2\frac{1-\cos(\|\mathbf{f}_{\omega}\|)}{\|\mathbf{f}_{\omega}\|^2}$

Problems

- Exponential mapping involves trascendental functions at each time step
- There exists conditions in which soft continuum robots does not undergone large deformations, but low deformations, even if yet finite.

Kinematics using the Cayley map

Cayley map: definition

$$egin{aligned} \mathsf{cay}(\widetilde{\mathbf{a}}) &= (\mathbf{I}+\widetilde{\mathbf{a}})(\mathbf{I}-\widetilde{\mathbf{a}})^{-1} = \ &= (\mathbf{I}-\widetilde{\mathbf{a}})^{-1}(\mathbf{I}+\widetilde{\mathbf{a}}) = \mathbf{A} \end{aligned}$$

- $A \in G$, Lie group
- $\bullet~\widetilde{a}\in\mathfrak{g},$ its corresponding Lie algebra
- I, identity matrix

Geometric interpretation

$$\exp(\widetilde{a}) \approx \mathsf{Pade}(\widetilde{a}) = (\mathsf{I} - \frac{1}{2}\widetilde{a})^{-1}(\mathsf{I} + \frac{1}{2}\widetilde{a})$$

• Cay map: Pade diagonal approximation of the exp map

Comparisons

... using the Exponential map

$$\mathbf{f} \in \mathbb{R}^{6} \mapsto \mathbf{H}(lpha) = \exp_{SE(3)}\left(lpha \mathbf{f}\right) \in SE(3)$$

•
$$\exp_{SE(3)}(\mathbf{f}) = \begin{bmatrix} \exp_{SO(3)}(\mathbf{f}_{\omega}) & \mathbf{T}_{SO(3)}^{T}(\mathbf{f}_{\omega})\mathbf{f}_{u} \\ \mathbf{0}_{1 \times 3} & 1 \end{bmatrix}$$

•
$$\exp_{SO(3)}(\mathbf{f}_{\omega}) = \mathbf{I}_{3\times 3} + \alpha(\mathbf{f}_{\omega})\widetilde{\mathbf{f}}_{\omega} + \frac{\beta(\mathbf{f}_{\omega})}{2}\widetilde{\mathbf{f}}_{\omega}^2$$

•
$$\mathbf{T}_{SO(3)}(\mathbf{f}_{\omega}) = \mathbf{I}_{3\times 3} - \frac{\beta(\mathbf{f}_{\omega})}{2}\widetilde{\mathbf{f}}_{\omega} + \frac{1-\alpha(\mathbf{f}_{\omega})}{\|\mathbf{f}_{\omega}\|^2}\widetilde{\mathbf{f}}_{\omega}^2$$

 $\alpha(\mathbf{f}_{\omega}) = \frac{\sin(\|\mathbf{f}_{\omega}\|)}{\|\mathbf{f}_{\omega}\|} \qquad \beta(\mathbf{f}_{\omega}) = 2\frac{1-\cos(\|\mathbf{f}_{\omega}\|)}{\|\mathbf{f}_{\omega}\|^2}$

... using the Cayley map

$$\mathbf{f} \in \mathbb{R}^{6} \mapsto \mathbf{M}(lpha) = \mathsf{cay}_{SE(3)}(lpha \mathbf{f}) \in SE(3)$$

$$\begin{array}{c} \bullet \ \mathsf{cay}_{SE(3)}(\mathbf{f}) = \\ \begin{bmatrix} \mathsf{cay}_{SO(3)}(\mathbf{f}_{\omega}) & \mathsf{dcay}_{SO(3)}(\mathbf{f}_{\omega})\mathbf{f}_{u} \\ \mathbf{0}_{1 \times 3} & 1 \end{bmatrix} \end{array}$$

•
$$\operatorname{cay}_{SO(3)}(\alpha \mathbf{f}_{\omega}) = I_{3\times3} + \frac{4\kappa}{4\kappa^2 + ||\mathbf{f}_{\omega}||^2} (\alpha \widetilde{\mathbf{f}}_{\omega} + \alpha \widetilde{\mathbf{f}}_{\omega}^2); \ k=1,1/2$$

•
$$dcay_{SO(3)}(\alpha \mathbf{f}_{\omega}) = cay_{SO(3)}(\alpha \mathbf{f}_{\omega}) + \mathbf{I}_{3 \times 3}$$

Difference between $\mathbf{H}(\alpha)$ and $\mathbf{M}(\alpha)$?

Comparisons (cont'd)

• Planar bending: $\mathbf{f} = \begin{bmatrix} 1 & 0 & 0 & 0 & K \end{bmatrix}^T$



Ten cases with increasing curvature. 1 : $K = \pi/10 \text{ m}^{-1}$;...; 10: $K = \pi \text{ m}^{-1}$

• Twist motion: $\mathbf{f} = \begin{bmatrix} 1 & 0 & 0 & \mathcal{T} & 0 & \mathcal{K} \end{bmatrix}^{\mathcal{T}}$



Five cases with increasing curvature and fixed torsion. 1 : $T = 3 \text{ m}^{-1}$ and K = 0; ...; 5: $T = 3 \text{ m}^{-1}$ and $K = 2\pi \text{ m}^{-1}$

Comparisons (cont'd)

- Geometric mechanics of soft robots
- Cayley transform for parametrization of soft robot kinematics, in case of low, yet finite deformations → advantages also for <u>compound robots</u>; <u>finite element models</u> using Cayley map as shape function
- Cayley map for recovering the geometrically exact motion, i.e. $\exp_{SE(3)}(\tilde{f}) = \exp_{SE(3)}(\tilde{\zeta})$ if we consider the change of deformation coordinates

$$\begin{bmatrix} \boldsymbol{\zeta}_{u} \\ \boldsymbol{\zeta}_{\omega} \end{bmatrix} = \begin{bmatrix} \lambda \boldsymbol{I}_{3\times3} & \tau \lambda' \boldsymbol{I}_{3\times3} \\ \boldsymbol{0}_{3\times3} & \lambda \boldsymbol{I}_{3\times3} \end{bmatrix} \begin{bmatrix} \boldsymbol{f}_{u} \\ \boldsymbol{f}_{\omega} \end{bmatrix}$$

with

$$\lambda = 2\kappa \frac{\tan(||\boldsymbol{f}_{\omega}||/2)}{||\boldsymbol{f}_{\omega}||}$$
$$\tau = \boldsymbol{u}\boldsymbol{f}_{\omega} = \boldsymbol{T}_{SO(3)}^{T}(\boldsymbol{f}_{\omega})\boldsymbol{f}_{u}\boldsymbol{f}_{\omega}$$

SimSOFT: a finite element solver for soft robots dynamics

SimSOFT

Input

- Geometry description and behavior (rigid / soft)
- Joints
- Actuation sources
- Planner
- Boundary conditions
- Solving parameters

Output

- Positions, velocities and accelerations of nodes and joints
- Forces and torques at boundaries and joint locations
- Stresses and strains of the elements
- Animations

Soft articulated robot composed by soft continuum elements











Conclusions

- Geometric approach for modeling soft robots
- Soft continuum robots modeled as Cosserat rod elements
- The deformation space formulation for soft robots dynamics
- The special case of constant curvature
- An alternative approach based on Cayley map
- A finite element solver for soft robots dynamics and relative applications

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